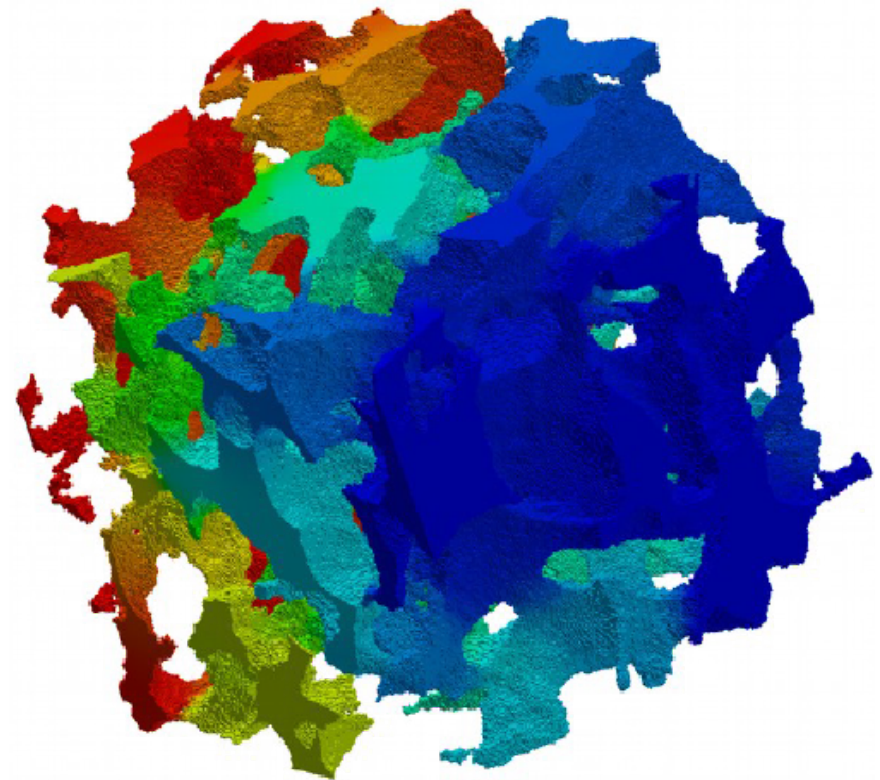


SIMULATION TECHNIQUES TO MODEL FLOW AND TRANSPORT AT THE PORE- SCALE

4TH CARGESE SUMMER SCHOOL
CARGESE, JULY 4, 2018

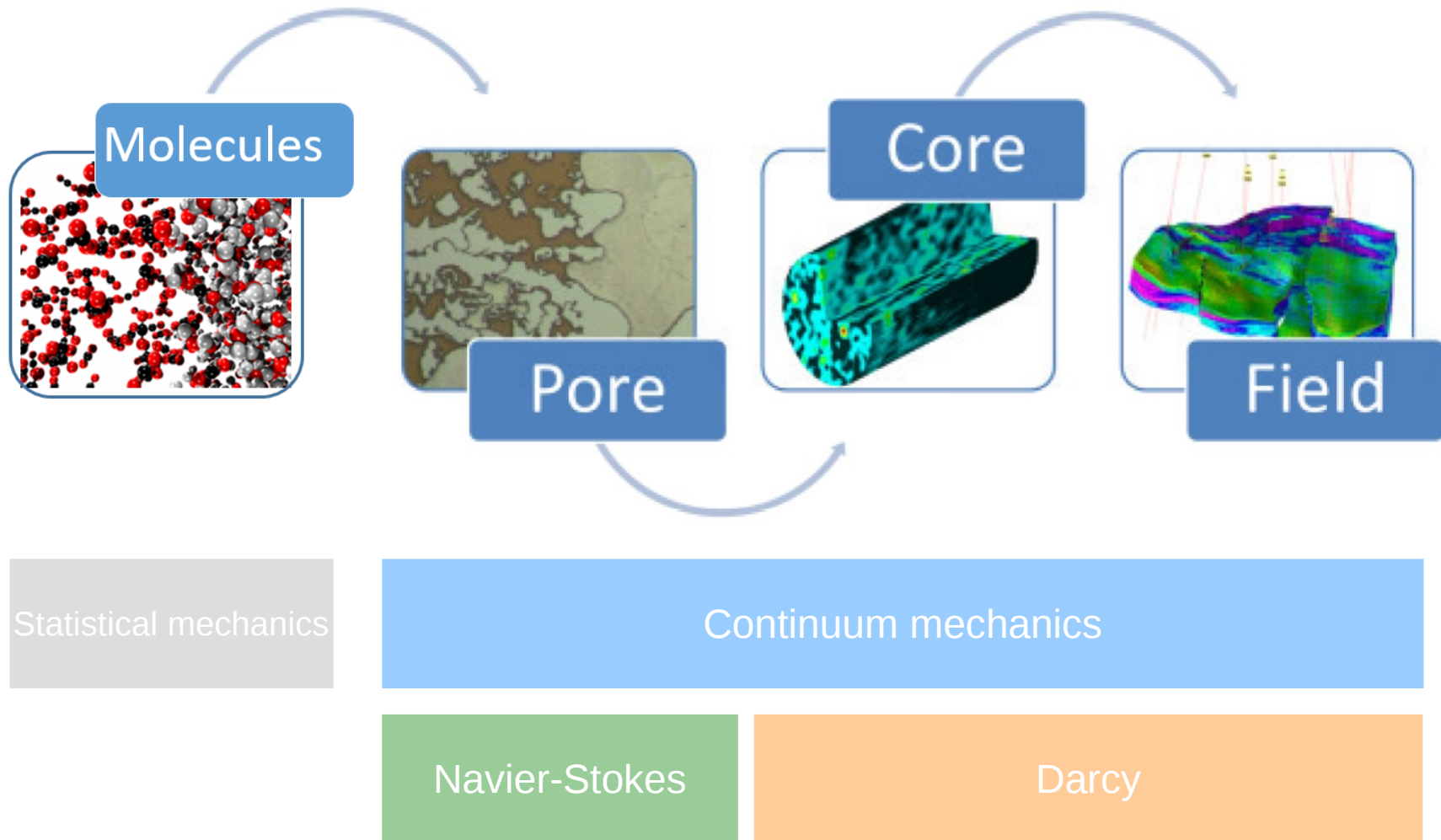
Cyprien Soulaine



Outline

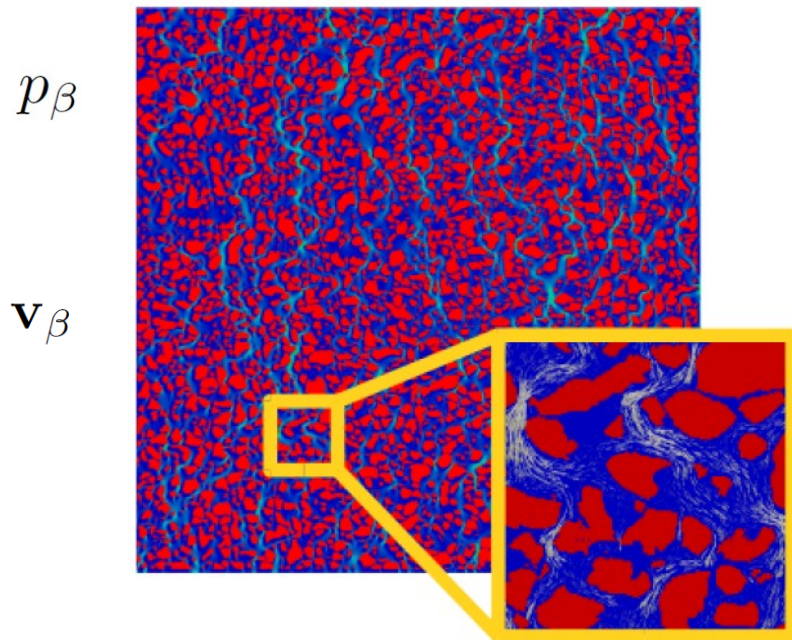
- Pore-scale modeling,
- Derivation of Navier-Stokes equations,
- Properties of Navier-Stokes equations,
- Numerical approaches to solve the flow at the pore-scale,
- Simulation examples,
- Two-phase flow at the pore-scale.

Multi-scale modeling

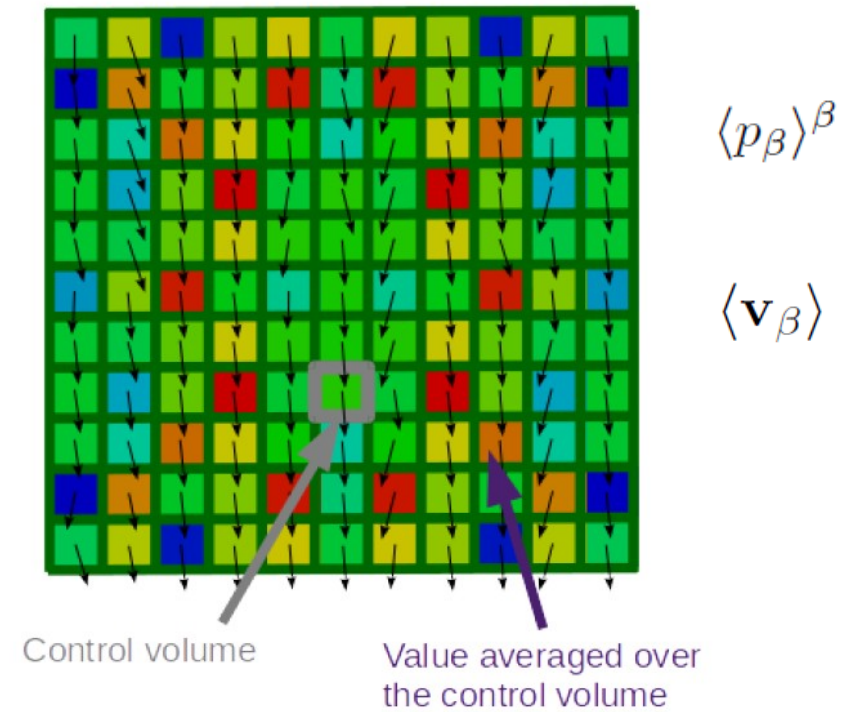


Discret vs continuum

direct modeling



continuum modeling



for every point of the domain

fluid **OR** solid

fluid **AND** solid

Digital Rock Physics

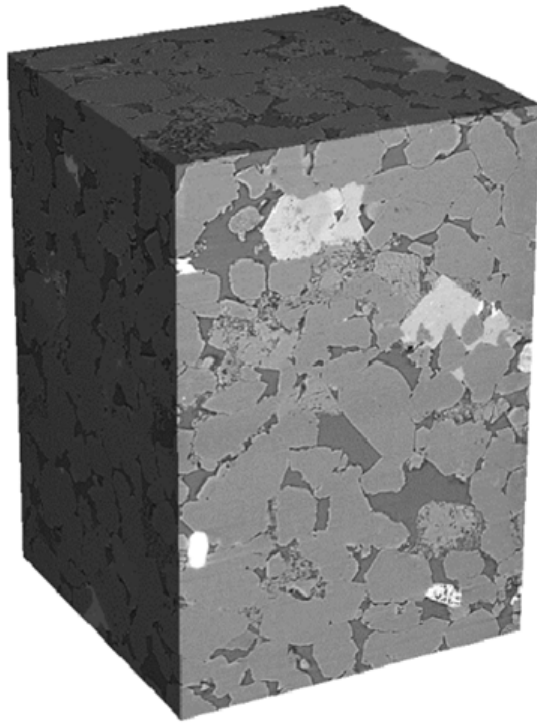
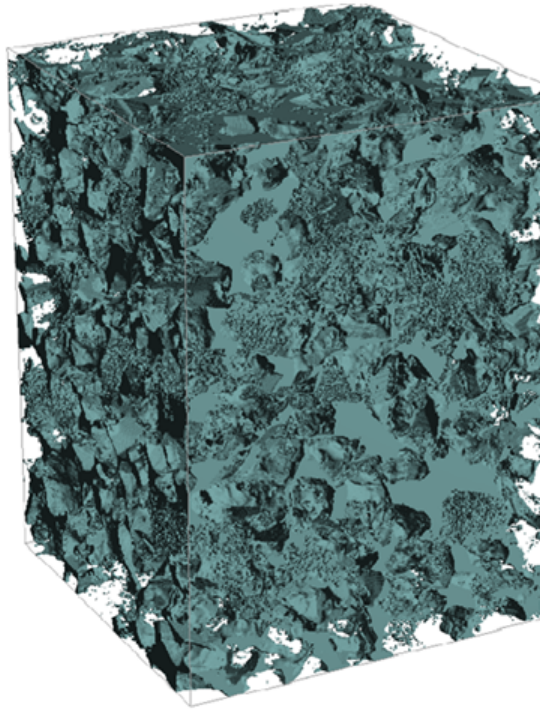
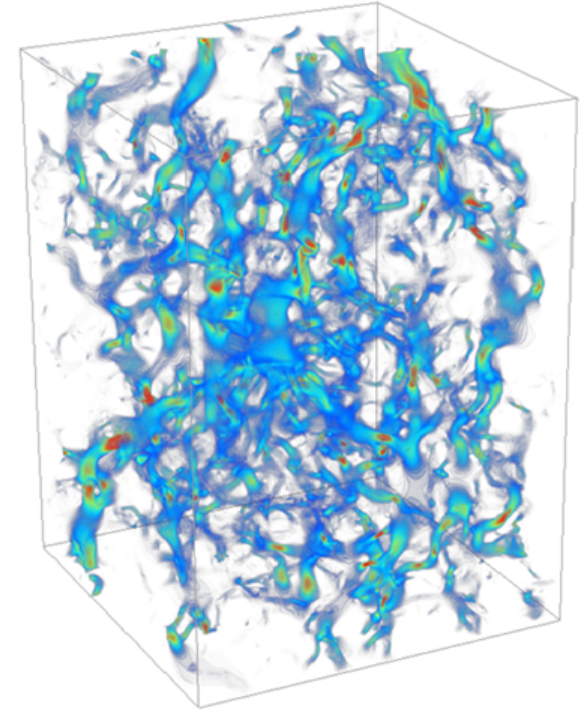


Image of core, plug or cutting



Segmented pores/minerals in image



Computation of rock properties

(source: GeoDict)

Geometrical parameters

- Porosity
- Percolation
- Surface area
- Tortuosity

Flow Parameters

- Permeability
- Multi-scale/phase flow
- Capillary pressure curve

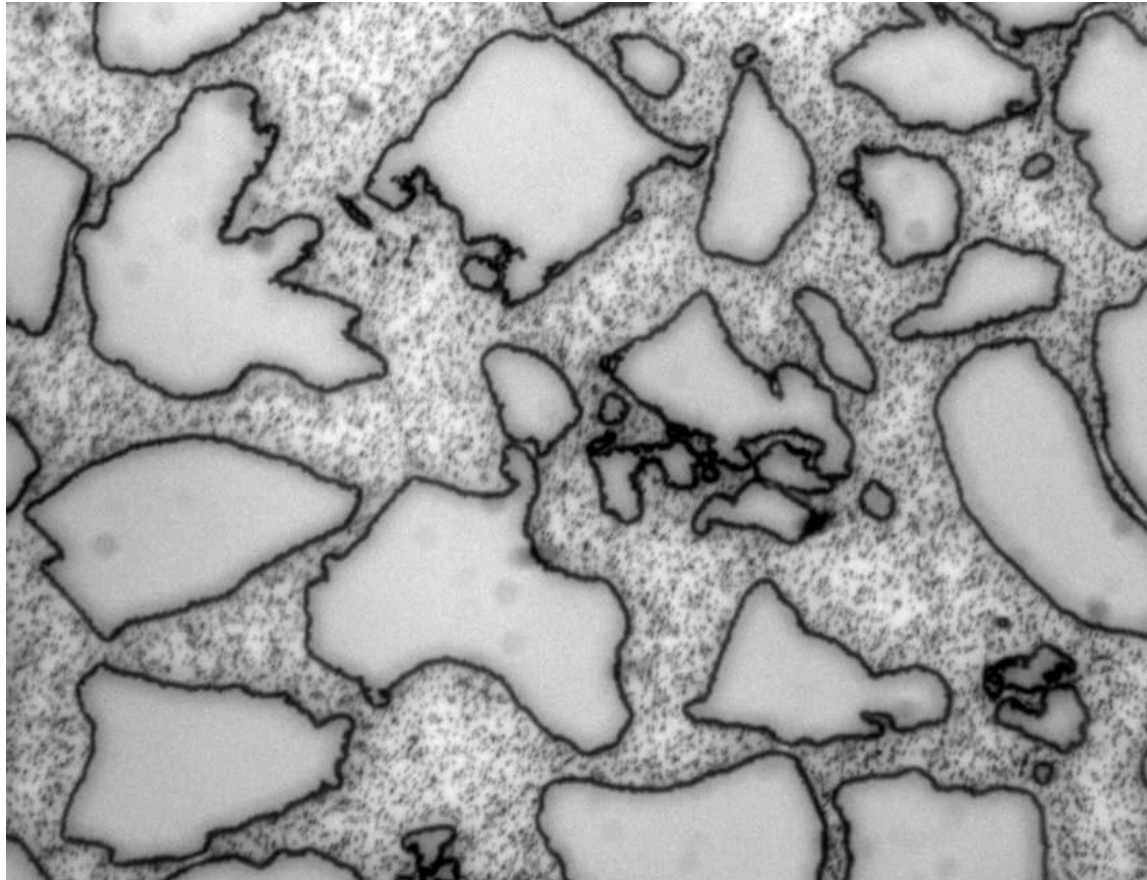
Electrical parameters

- Formation factor
- Resistivity index
- Saturation exponent
- Cementation exponent

Mechanical parameters

- Elastic moduli
- Stiffness
- In-Situ conditions

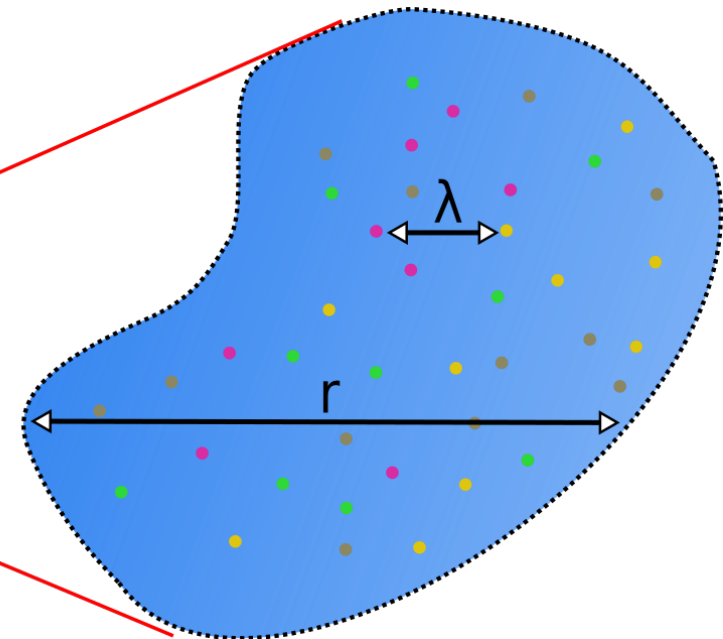
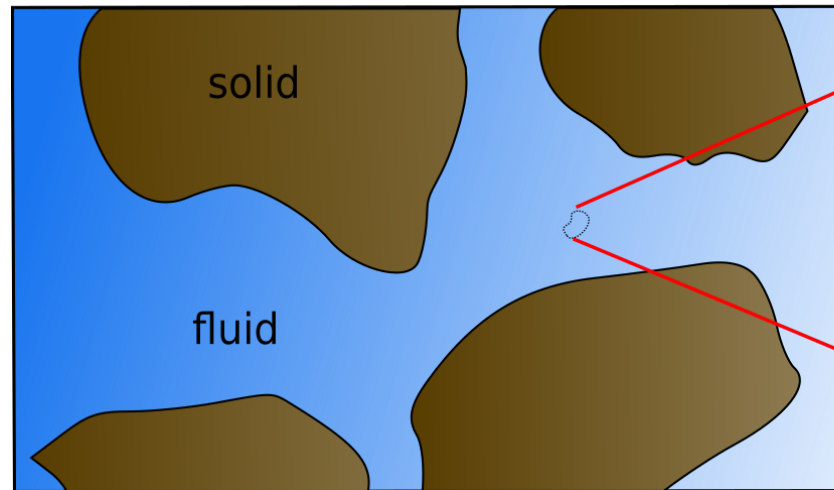
Single-phase flows at the pore-scale



- Water seeded with micro-particles to visualize the flow in the pore-space*,
- The particle trajectories are not random and are deterministic.

The Navier-Stokes equations

- The flow motion obeys the continuum mechanics conservation laws for fluid,
- These laws are derived from mass and momentum balance in a control volume,

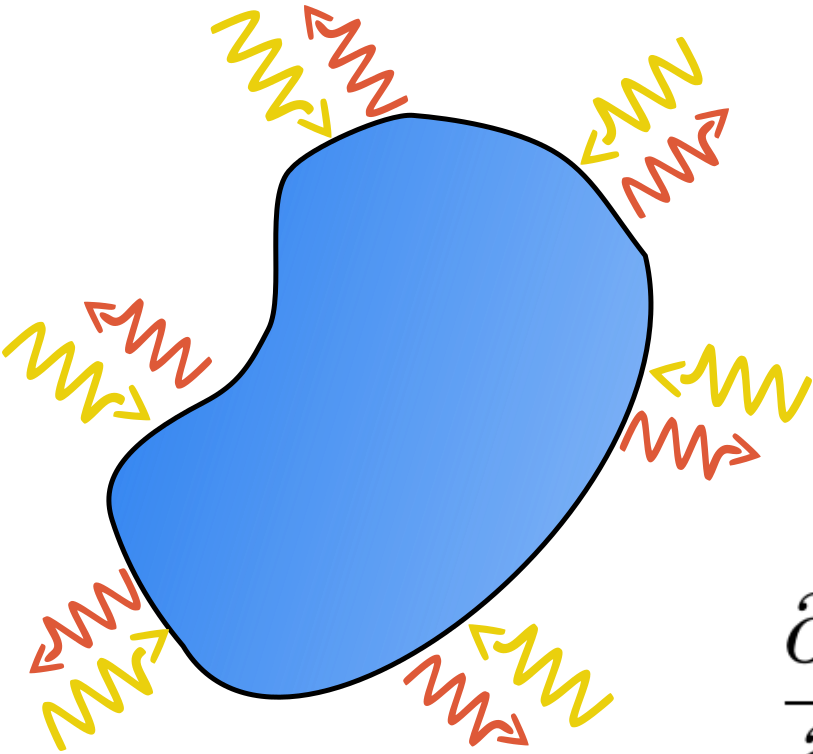


- The Navier-Stokes equations are made of:
 - A continuity equation,
 - A momentum equation,
 - Boundary conditions at the solid surface.

λ = mean free path

$\lambda \ll r$

Mass balance equation



$$\rho = \text{fluid density} = \frac{\text{mass of fluid}}{\text{control volume}}$$

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

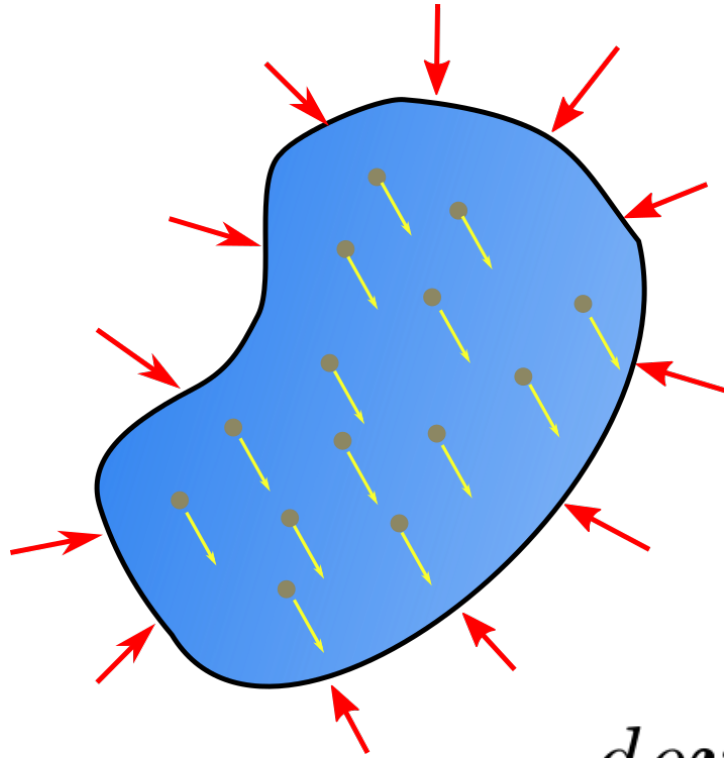
Variation of mass in
the control volume

Mass fluxes coming and
going out the control volume

For an incompressible fluid (most of
the liquids):

$$\nabla \cdot \mathbf{v} = 0$$

Cauchy momentum equation



$\rho \mathbf{v}$ = fluid momentum

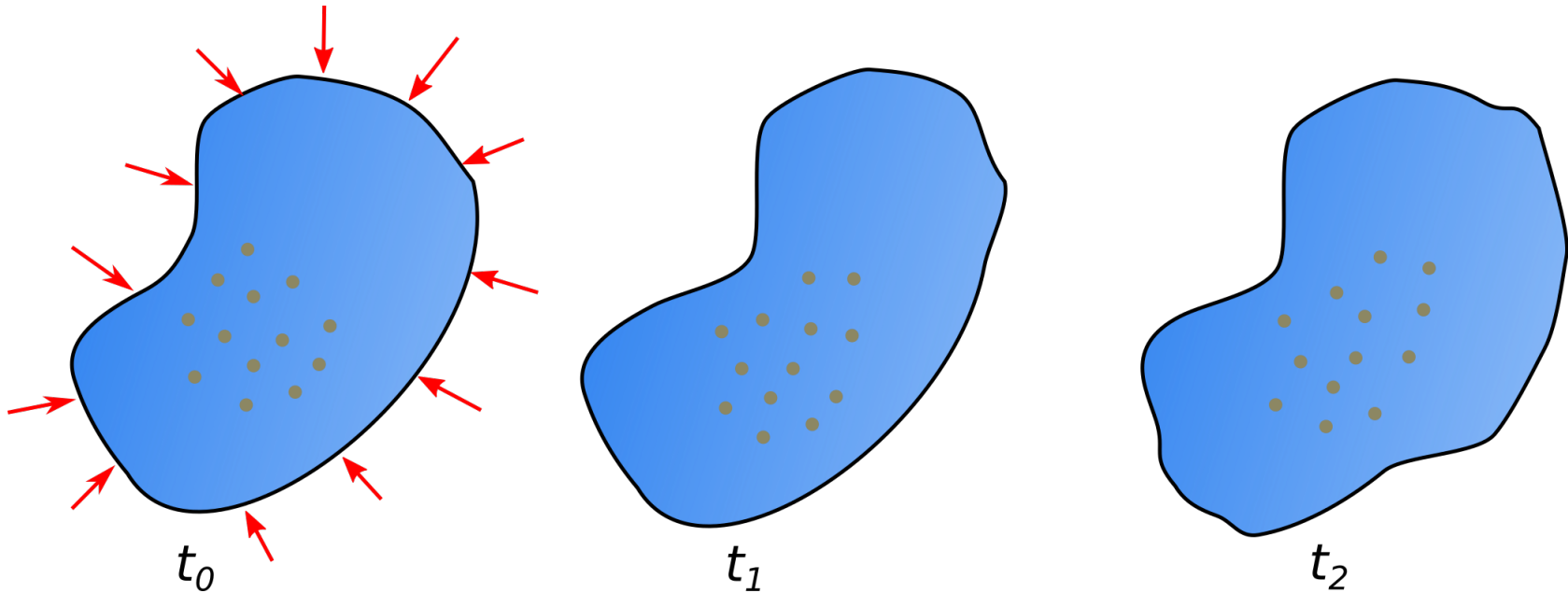
Conservation of momentum obtained from Newton's second law of motion (the variation of momentum and the sum of forces balance each other):

$$\frac{d\rho \mathbf{v}}{dt} = \underbrace{\nabla \cdot \boldsymbol{\sigma}}_{\text{surface forces}} + \underbrace{\rho \mathbf{f}}_{\text{body forces}}$$

Body forces = gravity, magnetic or electric fields...

Surface forces = forces responsible for the deformation of the control volume

Shear stress for Newtonian fluids



Cauchy stress tensor quantifies the change in shape and/or size of the control volume,

$$\underbrace{\sigma}_{\text{Cauchy stress tensor}} = \underbrace{-pI}_{\text{pressure forces}} + \underbrace{\tau}_{\text{viscous stress tensor}}$$

For Newtonian fluid, the viscous stress is proportional to the strain rate (the rate at which the fluid is being deformed),

$$\tau = \mu (\nabla \mathbf{v} + {}^t \nabla \mathbf{v})$$

Navier-Stokes momentum equation

In an Eulerian frame, the combination of Cauchy's law and Cauchy stress tensor gives the Navier-Stokes momentum equation,

$$\underbrace{\rho \frac{\partial \mathbf{v}}{\partial t} + \rho \mathbf{v} \cdot \nabla \mathbf{v}}_{\text{inertia}} = \underbrace{-\nabla p}_{\text{pressure gradient}} + \underbrace{\rho \mathbf{g}}_{\text{gravity}} + \underbrace{\mu \nabla^2 \mathbf{v}}_{\text{viscous force}}$$

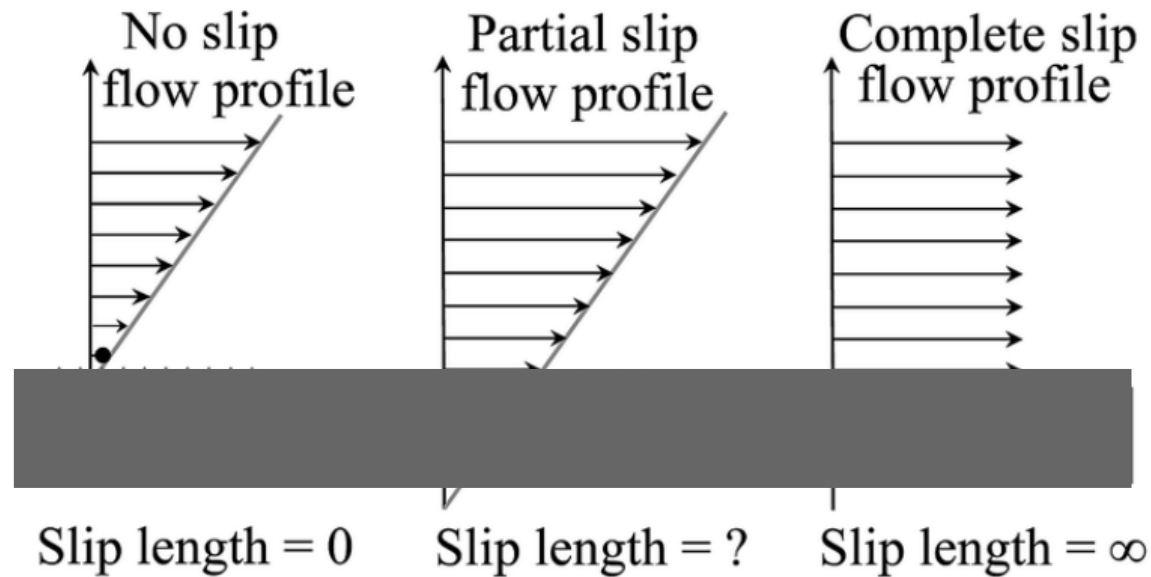
For slow flow, the inertial term is negligible compared with the dissipative viscous force and Navier-Stokes becomes the Stokes equation,

$$0 = -\nabla p + \rho \mathbf{g} + \mu \nabla^2 \mathbf{v}$$

Stokes is a particular case of Navier-Stokes, which means that all Navier-Stokes solvers are also valid for Stokes without any modification!

Boundary conditions at the solid surface

$$v = \underbrace{v_{\perp}}_{\text{normal component}} + \underbrace{v_{\parallel}}_{\text{tangential component}}$$

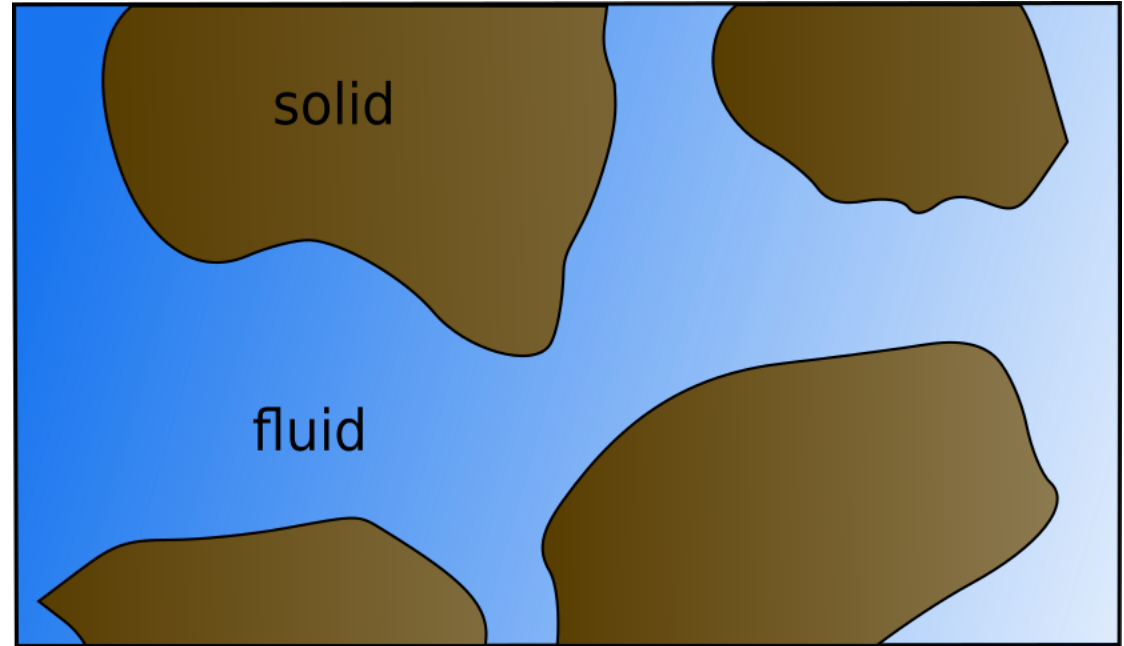


$$v = 0$$

$$v = \beta \frac{\partial v}{\partial n}$$

$$\frac{\partial v}{\partial n} = 0$$

The Navier-Stokes equations: summary



- Mass balance equation,

$$\nabla \cdot \mathbf{v} = 0$$

- Momentum balance equation,

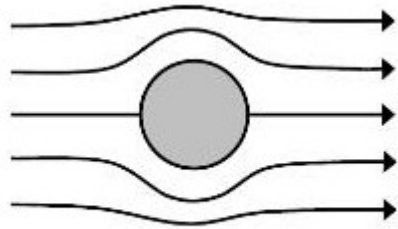
$$\rho \frac{\partial \mathbf{v}}{\partial t} + \rho \mathbf{v} \cdot \nabla \mathbf{v} = -\nabla p + \rho \mathbf{g} + \mu \nabla^2 \mathbf{v}$$

- No slip condition at the solid surface,

$$\mathbf{v} = 0$$

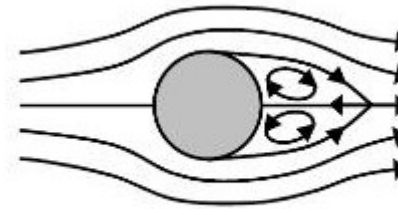
Different flow regimes (past a cylinder)

Creeping flow regime ($Re < 1$): the flow is governed by the viscous effects only and the streamlines embrace the solid structure. The flow is modeled by the Stokes equations. **Most of the time, we are in this situation.**



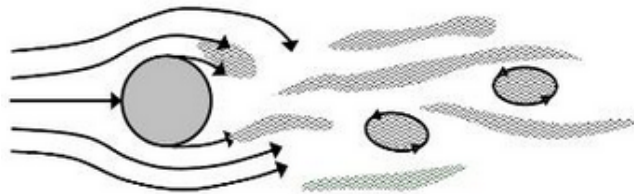
a

The inertia effects distort the streamlines and flow recirculations are generated downstream the obstacles. The flow is modeled by the laminar Navier-Stokes equations.



b

At very high Reynolds numbers, turbulence effects emerge. The flow is modeled by turbulent Navier-Stokes equations (RANS, LES...)

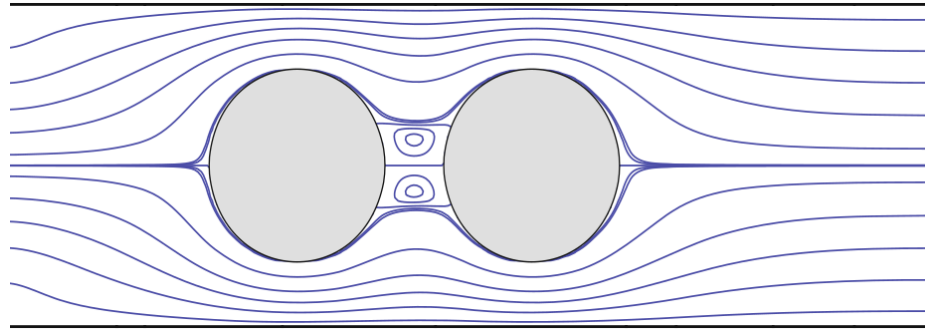


$$Re = \frac{\rho U_0 L_0}{\mu}$$

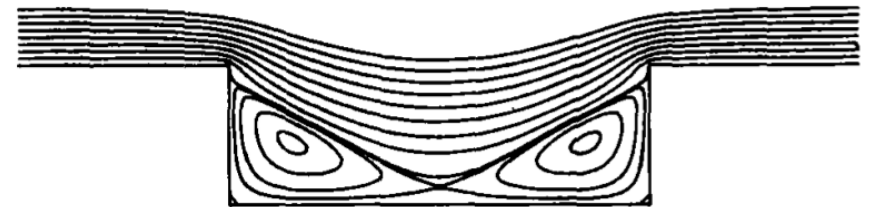
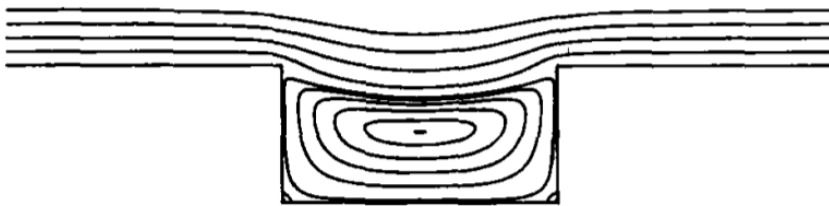


Recirculating motion in Stokes flows

- Space between two grains¹



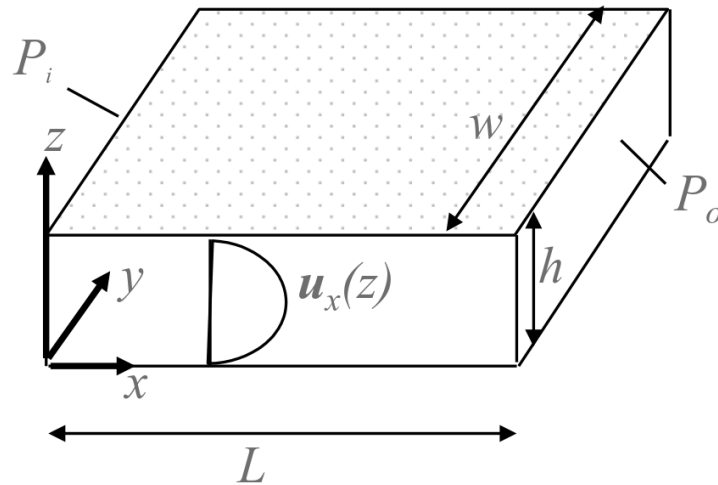
- Cavities/ Roughness²



¹Caltagirone, *Physique des écoulements continus*, 2013

²Higdon, *Stokes flow in arbitrary two-dimensional domains: shear flow over ridges and cavities*, Journal of Fluid Mechanics, 1985, 159, 195-226

Analytical solution: flow between two parallel plates



W and L assume to be large enough so

$$\mathbf{v} = v_x(z) \mathbf{e}_x$$

$$\nabla p = \frac{\Delta P}{L} \mathbf{e}_x.$$

The Stokes problem in Cartesian coordinates reads

$$\begin{cases} \mu \frac{\partial^2 v_x}{\partial z^2} = \frac{\Delta P}{L} \\ v_x \left(z = \pm \frac{h}{2} \right) = 0. \end{cases}$$



$$v_x(z) = \frac{1}{\mu} \frac{h^2}{8} \frac{\Delta P}{L} \left[1 - \left(\frac{2z}{h} \right)^2 \right]$$

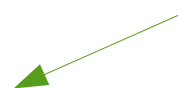
Parabolic profile



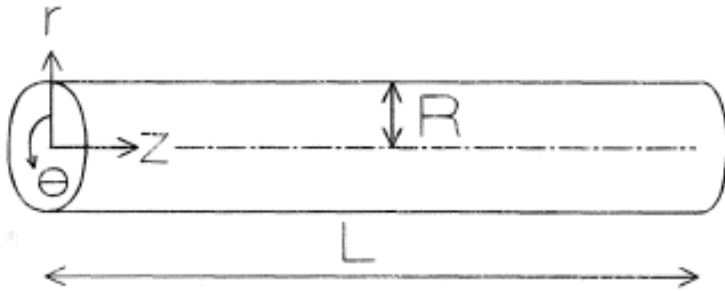
When volume averaging the velocity profile:

$$\bar{v}_x^f = \frac{1}{h} \int_{-\frac{h}{2}}^{\frac{h}{2}} v_x(z) dz = \frac{h^2}{12\mu} \frac{\Delta P}{L}$$

Hele-Shaw equation



Analytical solution: flow in a micro-tube



Due to the geometry, it is natural to use cylindrical coordinates.

L assumes to be long enough so the flow does not depend on z . Moreover, it is invariant by rotation and is carried by z only:

$$\mathbf{v} = v_z(r) \mathbf{e}_z \quad \nabla p = \frac{\Delta P}{L} \mathbf{e}_z$$

The Stokes problem in cylindrical coordinates reads

$$\left\{ \begin{array}{l} \mu \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) = \frac{\Delta P}{L} \\ v_z(r = R) = 0. \end{array} \right.$$



$$v_z(r) = \frac{R^2}{4\mu} \left(1 - \left(\frac{r}{R} \right)^2 \right) \frac{\Delta P}{L}$$

Parabolic profile

When volume averaging the velocity profile:

$$\bar{v}_z^f = \frac{2}{R^2} \int_0^R v_z(r) r dr = \frac{R^2}{8\mu} \frac{\Delta P}{L}$$

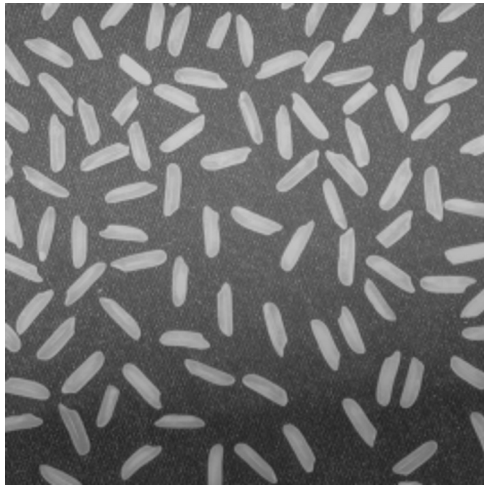


Hagen-Poiseuille law:

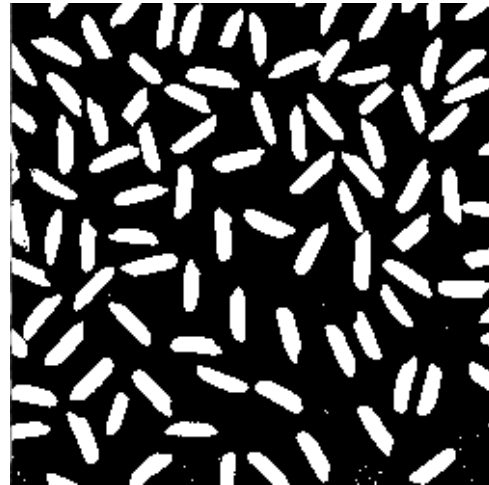
$$Q^f = \rho \bar{v}_z^f S = \frac{\pi R^4}{8\mu} \frac{\Delta P}{L}$$

How to solve flows in complex porous structures?

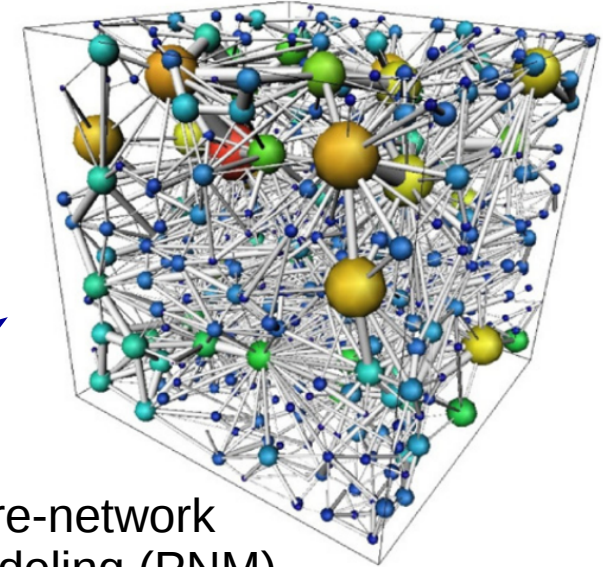
Source: Blunt et al. (2013)



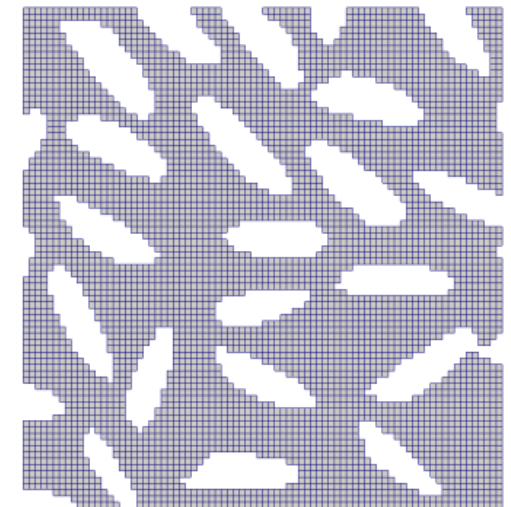
Acquisition of the
image (micro-CT ...)



Segmentation in fluid
and solid

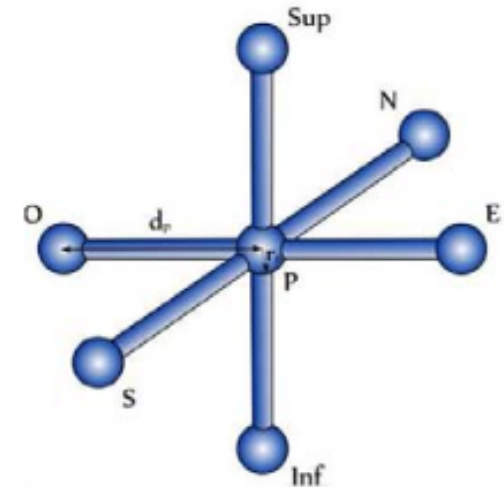
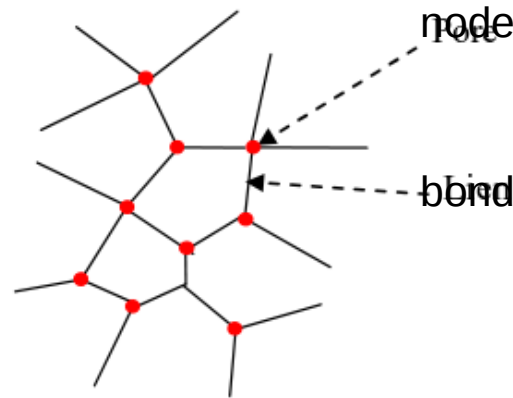
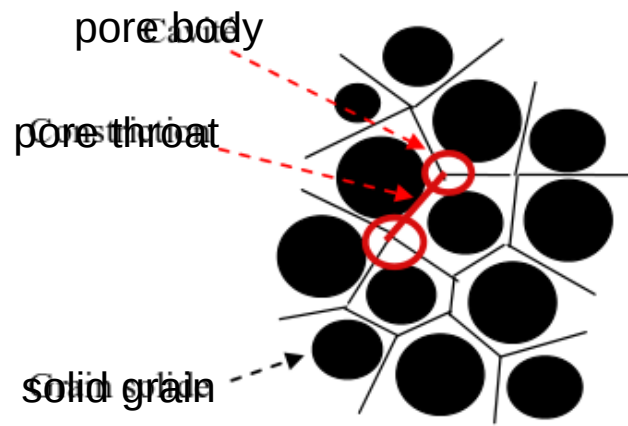


Pore-network
modeling (PNM)



Direct Numerical
Simulation (DNS)

Pore Network Modeling



For a cylindrical bond of radius r_i and length L_i that relates the nodes P and i , the mass flow rate is

$$Q_{ij} = g_{ij} (P_i - P_j) \quad \text{with} \quad g_{ij} = \frac{\rho \pi r_{ij}^4}{8 \mu L_{ij}}$$

Due to the mass conservation, for all pores P :

(cf. Poiseuille flow in a microtube)

$$\sum_j Q_{ij} = 0 \quad \rightarrow \quad \sum_j g_{ij} (P_i - P_j) = 0$$

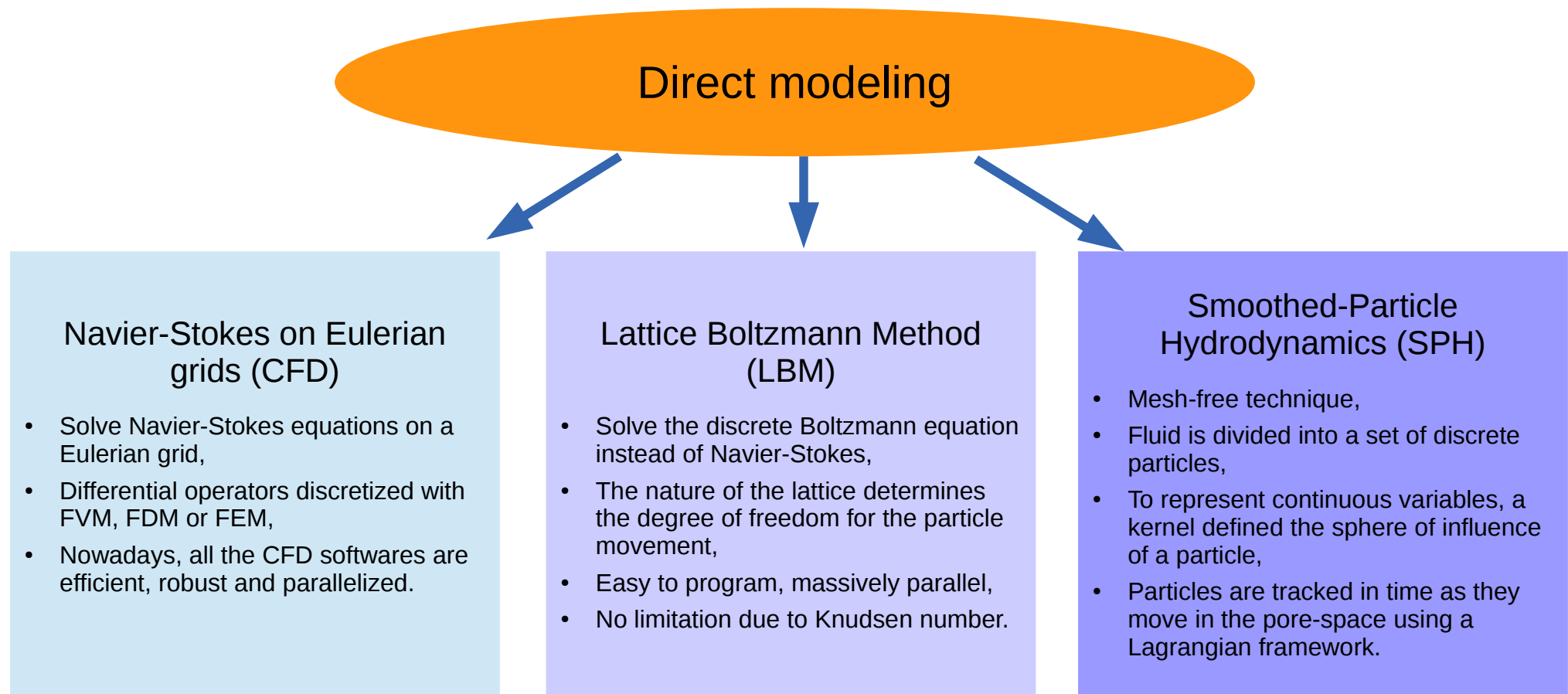


- Very easy to program
- Can compute relatively large domains (up to cm)



- Need more sophisticated models to be more representative of the actual geometry (bonds are not always cylindrical).
- It is not a tool to investigate the physics, the results will depend on the input...
- Still needs some research to the extension for multiphase

Direct Numerical Simulation techniques



- Directly deal with the real pore structure geometry,
- Can be used to investigate the physics



- More computationally expensive than PNM,
- Efficient multiphase solver are still in development.

Some popular CFD softwares



ANSYS Fluent is a CFD software using the finite volume method.



Another commercial CFD software.



COMSOL Multiphysics is a finite element package for various physics and engineering application, especially coupled phenomena or multiphysics.



OpenFOAM is a general purpose open-source CFD code. OpenFOAM is written in C++ and uses an object oriented approach which makes it easy to extend. The package includes modules for a wide range of applications. It uses the finite volume method.

<https://www.cypriensoulaine.com/openfoam>

Pressure-velocity coupling with projection algorithms

- Two equations, two unknowns (pressure and velocity) but no equation for the pressure field.

$$\nabla \cdot \mathbf{v} = 0 \quad \text{and} \quad \rho \frac{\partial \mathbf{v}}{\partial t} + \rho \mathbf{v} \cdot \nabla \mathbf{v} = -\nabla p + \rho \mathbf{g} + \mu \nabla^2 \mathbf{v}$$

- A pressure equation can be derived by taking the divergence of the momentum equation.
- The pressure and velocity equations are solved in a sequential manner using predictor-corrector projection algorithms. In OpenFOAM®, you have to choose between **PISO** and **SIMPLE**:

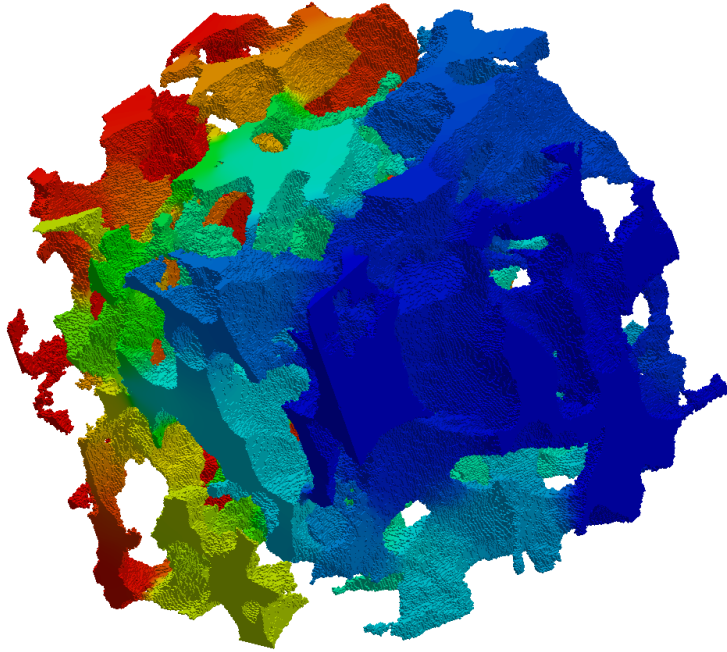
algorithm	transient	Steady-state	comments	OpenFOAM solver
PISO* PIMPLE	YES	YES	Can be used to find the stationary solution by solving all the time steps	icoFoam , pisoFoam, pimpleFoam, interFoam, twoPhaseEulerFoam, rhoPimpleFoam...
SIMPLE**	NO	YES	Faster than PISO to converge to the steady state	simpleFoam , rhoSimpleFoam...

- PISO** is not unconditionally stable and the time step is limited by a CFL condition.
- SIMPLE** is an iterative procedure that under-relaxes the pressure field and velocity matrix at each iteration.
- To allow larger time steps, a combination of both algorithm is sometime proposed (PIMPLE).
- Multiphase Navier-Stokes equations are solved in the framework of the **PISO** solution procedure.
- Stokes momentum equation does not involve transient terms and can be solved with **SIMPLE**.

* Issa. *Solution of the Implicitly Discretised Fluid Flow Equations by Operator-Splitting*. Journal of Computational Physics, 62:40-65, 1985.

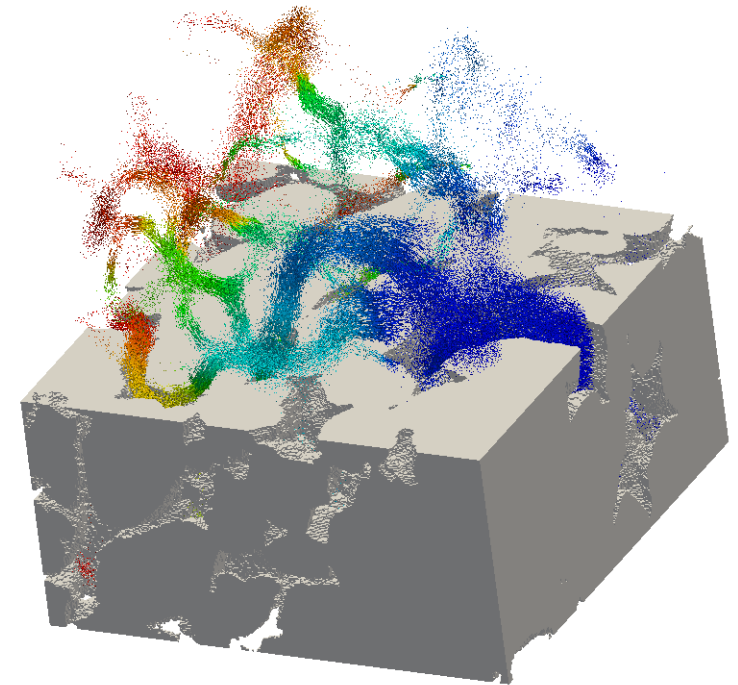
** Patankar. *Numerical Heat Transfer And Fluid Flow*, Taylor & Francis, 1980

Application: compute the permeability of a sandstone



$$K_{ij} = \mu \langle v_i \rangle \left(\frac{\Delta P}{L} \right)^{-1} \quad i = x, y, z$$

- Digital rock obtained from microtomography imaging,
- Grid the pore-space,
- In CFD simulations, the results may be very sensitive to the grid quality. At least 10 cells are required in each pore-throat,
- The grid quality is even more important when dealing with multiphase flow (refinement near the walls),
- Solve steady-state Stokes equations (SIMPLE algorithm with OpenFOAM).

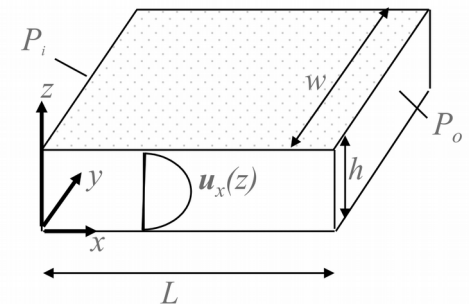
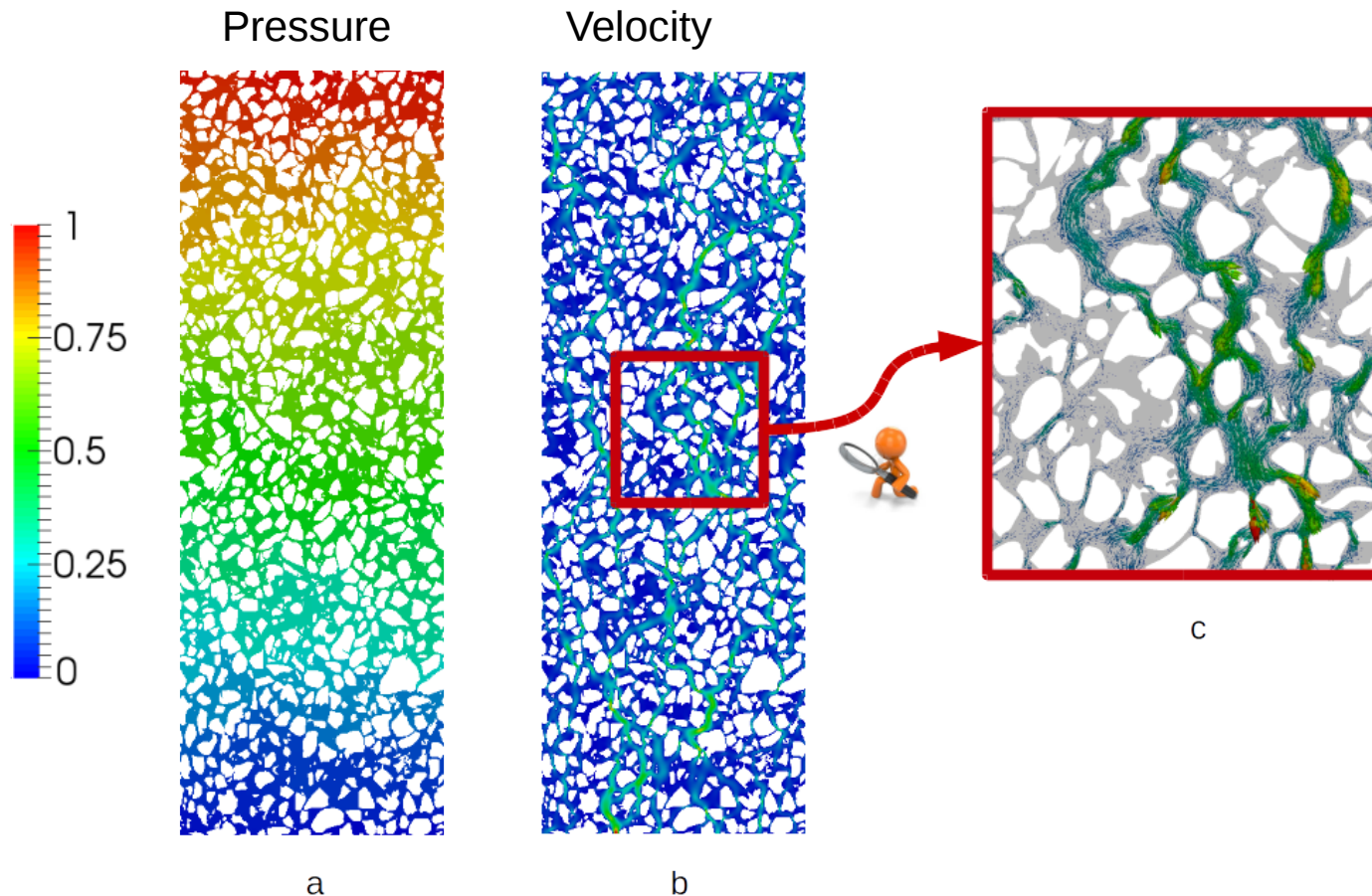


Simulations in “2.5D” micromodels

The simulation is 2D. The 3D effects are included assuming a Poiseuille profile in the depth of the micromodels and then considering Stokes flow equations averaged over the thickness, i.e. adding an Hele-Shaw correction term.

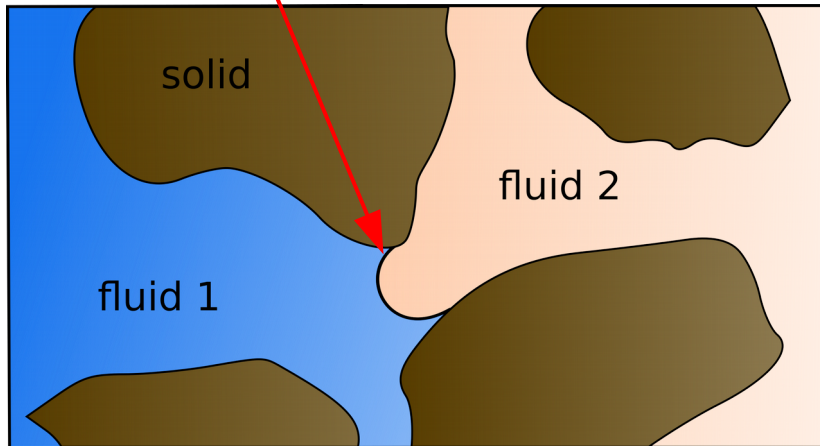
$$0 = -\nabla p + \mu \nabla^2 \mathbf{v} - \mu \frac{12}{h^2} \mathbf{v}$$

Hele-Shaw
correction term



Physics of two-phase flow in porous media at the pore-scale

Immiscible interface



Particularity of multi-phase flow

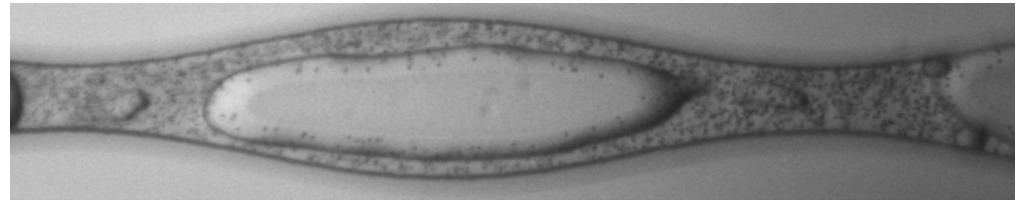
- Navier-Stokes equation in each phases
- Continuity of the tangential component of the velocity at the fluid/fluid interface
- Laplace law for a surface at the equilibrium

$$\Delta p = \sigma \left(\frac{1}{R_1} + \frac{1}{R_2} \right)$$

Surface tension
(N/m)

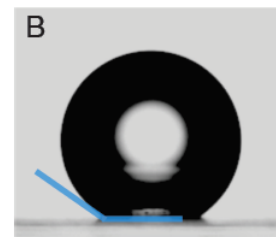
- Contact line dynamics at the solid surface

Surface tension is the elastic tendency of a fluid surface which makes it acquire the least surface area possible

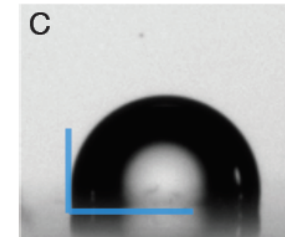


Sophie Roman (Univ of Orléans, FR)

The contact angle quantifies the wettability affinity of a solid surface by a liquid



$\theta=150^\circ$
non-wetting



$\theta=90^\circ$



$\theta=7^\circ$
wetting

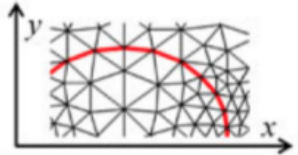
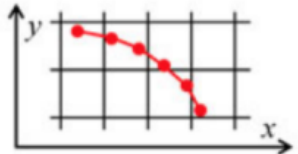
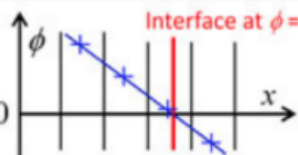
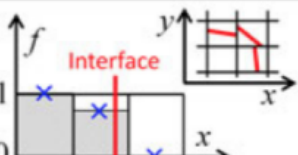
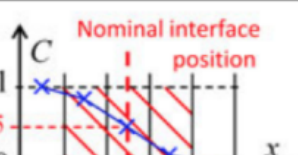
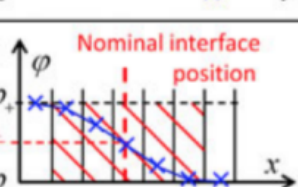
Zhao et al. (2016), PNAS

The displacement of a wetting fluid by a non-wetting fluid (drainage) is different than the displacement of a non-wetting fluid by a wetting fluid (imbibition)

CFD for two-phase flow

Simulation methods

Source: Wörner (2012)

	Interface representation	Interface evolution
Lagrangian type	Moving mesh 	Lagrangian movement of interface (unstructured grid)
	Front-tracking 	Lagrangian movement of interface marker points (•) within structured grid
Eulerian type	Level-set 	Advection equation for signed distance function ϕ
	IR-VOF 	Geometric evaluation of phase fluxes across mesh cell faces
	CF-VOF and C-LS 	Advection Eq. (11) for color function C (in C-LS followed by a compression step)
	Phase field 	Cahn-Hilliard Eq. (14) for order parameter ϕ

Challenges

- The state-of-the-art simulations **can not** go below $Ca=10^{-5}$. Otherwise spurious currents can pollute and even drive the interface propagation
- The physics of contact line dynamics is still poorly understood and can have an order one impact on the flow (Constant contact angle? Cox-Voinov model? Lubrication theory for thin film?).
- Solve the flow on large and complex domains.
- How to validate this numerical models?

The Volume of Fluid (VOF) technique

0	0	0	0	0	0
0	0	0	0	0	0
0	0.2	0.1	0	0	0.1
0.6	1	0.8	0.4	0.1	0.6
1	1	1	1	1	1
1	1	1	1	1	1

Color function

$$\alpha = \begin{cases} 0 & \text{in phase 2} \\ 0 < \alpha < 1 & \text{on the interface} \\ 1 & \text{in phase 1} \end{cases}$$

Single-field variables

$$\mathbf{v} = \alpha \mathbf{v}_1 + (1 - \alpha) \mathbf{v}_2$$

$$p = \alpha p_1 + (1 - \alpha) p_2$$

$$\rho = \alpha \rho_1 + (1 - \alpha) \rho_2$$

$$\mu = \alpha \mu_1 + (1 - \alpha) \mu_2$$

Single-field equations¹

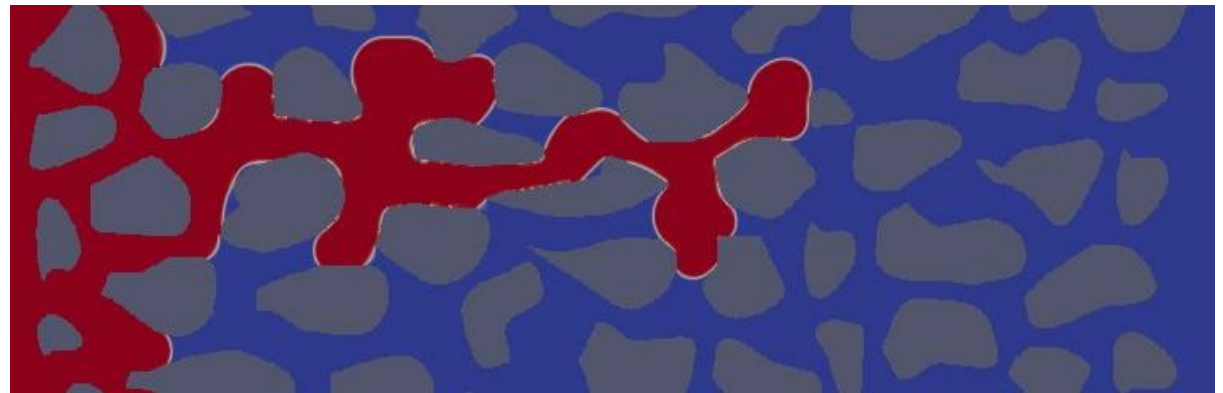
$$\rho \frac{\partial \mathbf{v}}{\partial t} + \rho \mathbf{v} \cdot \nabla \mathbf{v} = -\nabla p + \rho \mathbf{g} + \nabla \cdot \mu (\nabla \mathbf{v} + {}^t \nabla \mathbf{v}) + \mathbf{F}_c$$

$$\nabla \cdot \mathbf{v} = 0$$

$$\frac{\partial \alpha}{\partial t} + \mathbf{v} \cdot \nabla \alpha = 0$$

Continuum Surface Force (CSF)²

$$\mathbf{F}_c = \sigma \nabla \cdot \left(\frac{\nabla \alpha}{\|\nabla \alpha\|} \right) \nabla \alpha$$



Graveleau et al. 2017

¹Hirt, C. & Nichols, B. *Volume of fluid (VOF) method for the dynamics of free boundaries* Journal of Computational Physics, 1981, 39, 201 - 225

²Brackbill et al. *A continuum method for modeling surface tension* Journal of Computational Physics, 1992, 100, 335 - 354

Thank you for your
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